



#### **Course Outline**

- L1-L5 Specifications, models and loop-shaping by hand
- L6–L8 Limitations on achievable performance
  - Controllability/observability, multivariable poles/zeros
  - Fundamental limitations
  - Multivariable and decentralized control
- L9-L11 Controller optimization: analytic approach
- L12-L14 Controller optimization: numerical approach
  - L15 Course review



#### **L7: Fundamental Limitations**

- Bode's Relation
- 2 Limitations from RHP poles/zeros and delays
  - Insights and rules of thumb from loop shaping
  - Example: Rear-wheel steering bike
  - Hard proofs using the Maximum Modulus Principle
- Bode's Integral



## Limitations in control design

#### Some things we already know:

- Model uncertainty, measurement noise, and control signal limitations give upper limits on the achievable bandwidth
- S + T = 1, which implies

$$|S(i\omega)| + |T(i\omega)| \ge 1$$
  
 $||S(i\omega)| - |T(i\omega)|| \le 1$ 

- Some modes may be impossible to control or observe due to lack of controllability or observability
- A zero in the origin makes it impossible to control the process in stationarity



## Limitations in control design

Some **fundamental limitations** of linear control systems exist, regardless of the controller design:

- Bode's Relation: amplitude and phase are coupled
- Limitations from non-minimum-phase elements:
  - unstable poles
  - right-half-plane (RHP) zeros
  - time delays
- ullet Bode's Integral:  $|S(i\omega)|$  cannot be made small everywhere



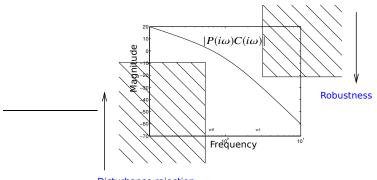
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## **Recall: Loop shaping**

The magnitude of the loop transfer function L=PC should be made large at low frequencies and small at high frequencies:

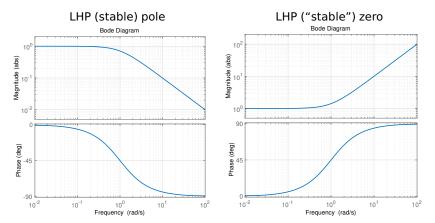


Disturbance rejection

How quickly can we make the transition from high to low gain and still retain a good phase margin?



# Amplitude and phase are coupled



If G(s) is minimum phase (no RHP poles/zeros or time delays) then

$$\arg G(i\omega) \approx \frac{\pi}{2} \frac{d \log |G(i\omega)|}{d \log \omega}$$



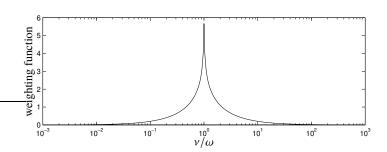
#### **Bode's Relation**

If G(s) is minimum phase, then

$$\arg G(i\omega) = \frac{2\omega}{\pi} \int_0^\infty \frac{\log|G(i\nu)| - \log|G(i\omega)|}{\nu^2 - \omega^2} d\nu$$

$$= \frac{1}{\pi} \int_0^\infty \frac{d\log|G(i\nu)|}{d\log\nu} \underbrace{\log\left|\frac{\nu + \omega}{\nu - \omega}\right|}_{\text{weighting function}} d\log\nu \approx \frac{\pi}{2} \frac{d\log|G(i\omega)|}{d\log\omega}$$

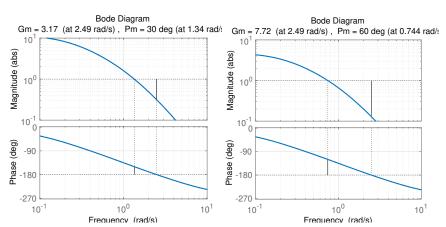
weighting function





## Consequence for phase margin

For minimum-phase systems, to have a phase margin between  $30^\circ$  and  $60^\circ$ , the slope of the amplitude curve should be between approx.  $-\frac{5}{3}$  and  $-\frac{4}{3}$  at the cross-over frequency.





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### Non-minimum-phase systems

A transfer function G(s) can always be factored as

$$G(s) = G_{mp}(s) G_{nmp}(s)$$

#### such that

- $G_{mp}(s)$  only contains minimum-phase elements
- $G_{nmp}(s)$  contains non-minimum-phase elements and has
  - unit magnitude:  $|G_{nmp}(i\omega)| = 1$
  - negative phase:  $\arg G_{nmp}(i\omega) < 0$



#### Non-minimum-phase elements

Pole in the right half-plane at p:

$$G_{nmp}(s) = \frac{s+p}{s-p}$$

Zero in the right half-plane at z:

$$G_{nmp}(s) = \frac{z - s}{s + z}$$

Time delay of length L:

$$G_{nmp}(s) = e^{-sL}$$

# Insights and rules of thumb from loop shaping

The minimum-phase part of the system can be shaped to our liking, to achieve a suitable cross-over frequency  $\omega_c$  and phase margin  $\varphi_m$ . However,

- An RHP pole p decreases the phase by  $> 90^{\circ}$  for  $\omega < p$ . To retain a reasonable phase margin, we must have  $\omega_c > p$ .
- An RHP zero z decreases the phase by  $> 90^{\circ}$  for  $\omega > z$ . To retain a reasonable phase margin, we must have  $\omega_c < z$ .
- A time delay L decreases the phase by  $\omega L$ . To retain a reasonable phase margin, we must have  $\omega_c < \frac{\pi/2}{L} \approx \frac{1.6}{L}$ .



## **Example: Rear-wheel steering bike**

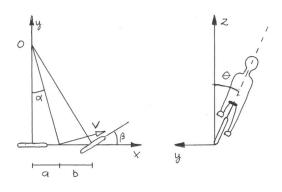






### **Bike example**

A (linearized) torque balance of a normal bike can be written as



$$J\frac{d^2\theta}{dt^2} = mg\ell\theta + \frac{mV_0\ell}{b} \left( V_0\beta + a\frac{d\beta}{dt} \right)$$



### Bike example, cont'd

$$J\frac{d^{2}\theta}{dt^{2}} = mg\ell\theta + \frac{mV_{0}\ell}{b} \left( V_{0}\beta + a\frac{d\beta}{dt} \right)$$

where the physical parameters have typical values as follows:

Mass: m = 70 kg

Distance rear-to-center: a = 0.3 m

Height over ground:  $\ell$  = 1.2 m

Distance center-to-front: b = 0.7 m

Moment of inertia:  $J = 120 \text{ kgm}^2$ 

Speed:  $V_0 = 5 \text{ ms}^{-1}$ 

Acceleration of gravity:  $g = 9.81 \text{ ms}^{-2}$ 

The transfer function from  $\beta$  to  $\theta$  is

$$P(s) = \frac{mV_0\ell}{b} \frac{as + V_0}{Js^2 - mg\ell}$$



### Bike example, cont'd

The system has an unstable pole at

$$p = \sqrt{\frac{mg\ell}{J}} \approx 2.6$$

The closed-loop system must be at least as fast as this. Moreover, the transfer function has a zero at

$$z = -\frac{V_0}{a} \approx -3.3V_0$$

For the rear-wheel steering bike we have the same pole but a minus sign before  $V_0$  and the zero will thus be in the RHP!

An unstable pole-zero cancellation occurs for  $V_0 \approx 0.8$  m/s.



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## Hard bounds from RHP poles/zeros

The sensitivity function must be 1 at a RHP zero z:

$$P(z) = 0$$
  $\Rightarrow$   $S(z) = \frac{1}{1 + \underbrace{P(z) C(z)}_{0}} = 1$ 

Similarly, the complementary sensitivity function must be 1 at an unstable pole p:

$$P(p) = \infty$$
  $\Rightarrow$   $T(p) = \frac{P(p)C(p)}{1 + P(p)C(p)} = 1$ 



## The Maximum Modulus principle

Suppose that all poles of the rational function G(s) have negative real part. Then

$$\sup_{\omega \in \mathbb{R}} |G(i\omega)| \ge |G(s)|$$

for all s in the right half-plane.

#### Hard bounds on specifications on S

#### THEOREM:

Given stable  $W_s(s)$  and  $S(s)=(1+L(s))^{-1}$ , the specification  $\|W_sS\|_\infty \leq 1$ 

can be met **only if**  $|W_s(z)| \le 1$  for every RHP zero z of L(s).

#### Proof

$$||W_s S||_{\infty} = \sup_{\omega \in \mathbb{R}} |W_s(i\omega)S(i\omega)| \ge |W_s(s)S(s)|$$

for all s in RHP. For s=z, the right hand side becomes  $|W_s(z)|$ , which in turn gives the necessary condition above.



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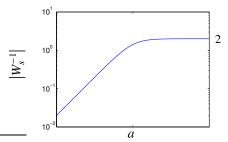
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## Example - hard bound from RHP zero

Assume the sensitivity specification  $W_s(s) = \frac{s+a}{2s}$ , a > 0.



If the plant has a RHP zero in z, then  $||W_sS||_{\infty} \le 1$  is impossible to fulfill unless

$$|W_s(z)| = \left| \frac{z+a}{2z} \right| \le 1 \quad \Leftrightarrow \quad a \le z$$

("Closed loop must be slower than z for reasonable robustness,  $M_s \leq 2$ ")



## Hard bounds on specifications on T

#### THEOREM:

Given stable 
$$W_t(s)$$
 and  $T(s)=(1+L(s))^{-1}L(s)$ , the specification 
$$\|W_tT\|_{\infty}\leq 1$$

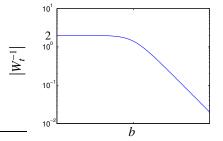
can be met **only if**  $|W_t(p)| \le 1$  for every RHP pole p of L(s).

(Proof is analogous to the one above)



## **Example – hard bound from unstable pole**

Assume the compl. sensitivity specification  $W_t = \frac{s+b}{2b}$ , b > 0



If the plant has an unstable pole in p, then  $||W_tT||_{\infty} \leq 1$  is impossible to fulfill unless

$$|W_t(p)| = \left| \frac{p+b}{2b} \right| \le 1 \quad \Leftrightarrow \quad b \ge p$$

("Closed loop must be faster than p for reasonable robustness,  $M_t \leq 2$ ")



#### RHP zero and unstable pole

For a system with both RHP zero  $\boldsymbol{z}$  and unstable pole  $\boldsymbol{p}$  it can be shown that

$$M_s = \sup_{\omega} |S(i\omega)| \ge \left| \frac{z+p}{z-p} \right|$$

(See lecture notes for details)

If  $p \approx z$  the sensitivity function must have a high peak **for every controller** C.

**Example:** Bicycle with rear-wheel steering

$$\frac{\theta(s)}{\delta(s)} = \frac{am\ell V_0}{bJ} \cdot \frac{(-s + V_0/a)}{(s^2 - mg\ell/J)}$$



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## **Bode's Integral – stable systems**

For a stable loop gain L(s) with relative degree  $\geq 2$  the following **conservation law** for the sensitivity function  $S(s) = (1 + L(s))^{-1}$  holds:

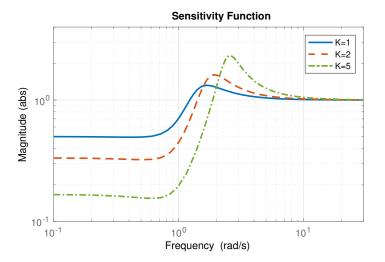
$$\int_0^\infty \log |S(i\omega)| d\omega = 0$$

(Sometimes known as the "waterbed effect")



# **Example**

Proportional control of  $(s^2 + s + 1)^{-1}$ 





#### **Bode's Integral - general case**

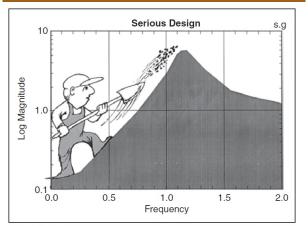
For a loop gain with relative degree  $\geq 2$  and unstable poles  $p_1, \ldots, p_M$ , the following **conservation law** for the sensitivity function holds:

$$\int_0^\infty \log |S(i\omega)| d\omega = \pi \sum_{i=1}^M \operatorname{Re}(p_i)$$

(There exists a similar condition relating T(s) and RHP zeros, see the lecture notes.)



#### G. Stein: "Conservation of dirt!"



**Figure 3.** Sensitivity reduction at low frequency unavoidably leads to sensitivity increase at higher frequencies.

Picture from Gunter Stein's Bode Lecture (1985) "Respect the unstable". Reprint in *IEEE Control Systems Magazine*, Aug 2003.



### **Lecture 7 – summary**

- Bode's Relation and Bode's Integral
- Limitations from unstable poles, RHP zeros and time delays
  - ullet Rules of thumb for achievable  $\omega_c$
- Limitations on specifications on *S* and *T* from RHP zeros and poles: Hard proofs using Maximum Modulus principle
- Example: Rear-wheel steering bicyle pole and zero i RHP